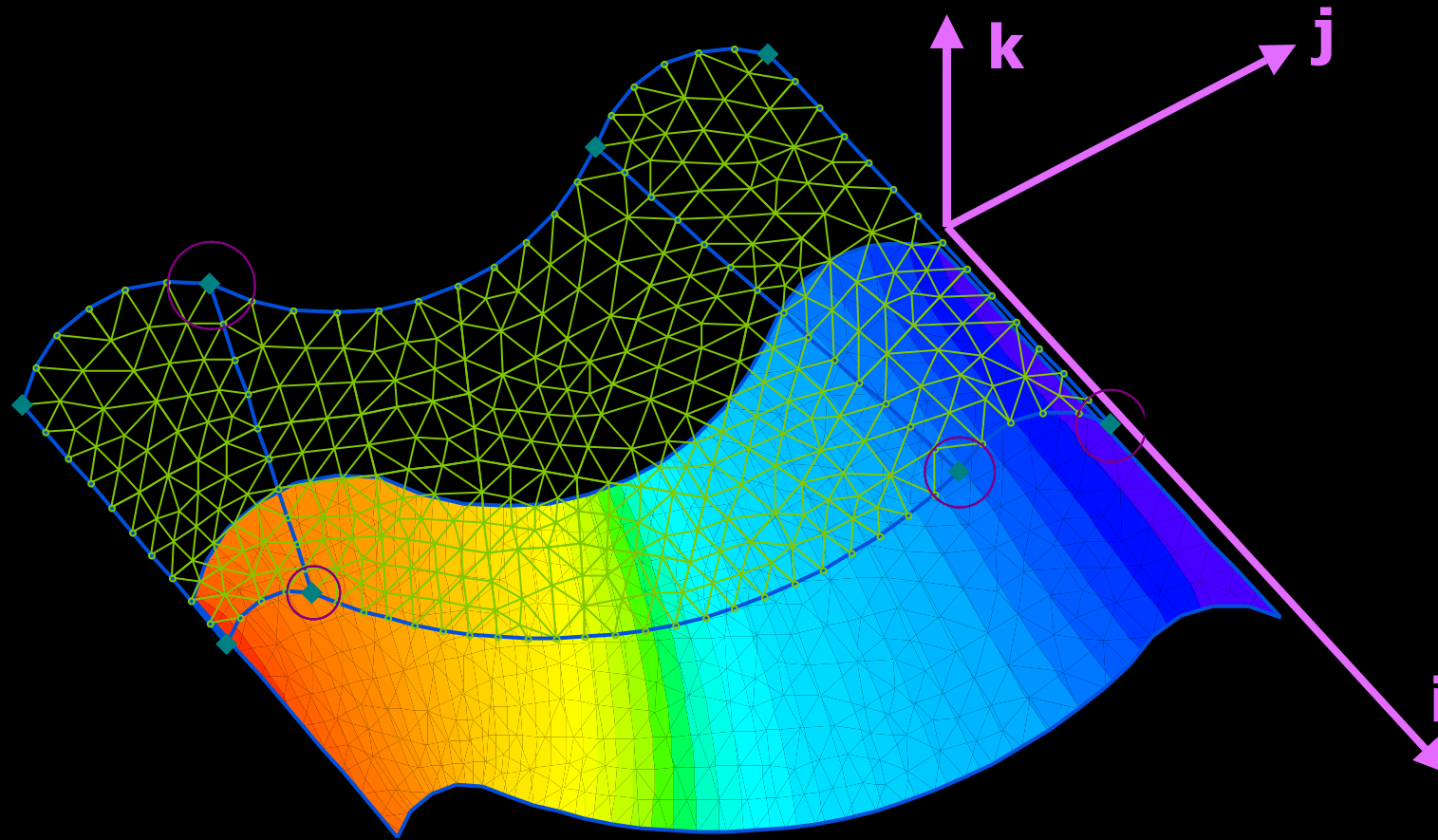


Application of QuickField Software to Heat Transfer Problems



By Dr. Evgeni Volpov

Basic Formulations for GIS HT model

Classical Heat Transfer Equations

Heat-transfer equation for linear problems is:

$$\frac{\partial}{\partial x} \left(\lambda_x \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left(\lambda_y \frac{\partial T}{\partial y} \right) = -q - c\rho \frac{\partial T}{\partial t} \quad \text{- planar case;}$$

$$\frac{1}{r} \frac{\partial}{\partial r} \left(\lambda_r r \frac{\partial T}{\partial r} \right) + \frac{\partial}{\partial z} \left(\lambda_z \frac{\partial T}{\partial z} \right) = -q - c\rho \frac{\partial T}{\partial t} \quad \text{- axisymmetric case;}$$

for nonlinear problems:

$$\frac{\partial}{\partial x} \left(\lambda(T) \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left(\lambda(T) \frac{\partial T}{\partial y} \right) = -q(T) - c(T)\rho \frac{\partial T}{\partial t} \quad \text{- planar case;}$$

$$\frac{1}{r} \frac{\partial}{\partial r} \left(\lambda(T) r \frac{\partial T}{\partial r} \right) + \frac{\partial}{\partial z} \left(\lambda(T) \frac{\partial T}{\partial z} \right) = -q(T) - c(T)\rho \frac{\partial T}{\partial t} \quad \text{- axisymmetric case;}$$

where:

T - temperature;

t - time;

$\lambda_{x(y,r,z)}$ - components of heat conductivity tensor;

$\lambda(T)$ - heat conductivity as a function of temperature approximated by cubic spline (anisotropy is not supported in nonlinear case);

$q(T)$ - volume power of heat sources, in linear case - constant, in nonlinear case - function of temperature approximated by cubic spline;

$c(T)$ - specific heat, in nonlinear case - function of temperature approximated by cubic spline;

ρ - density of the substance.

In linear case all the parameters are constants within each block of the model.

Boundary Conditions

1. $T(S) = T_0$ Const Temperature

$T(S) = T_0 + k.S$ Linear Temp.

2. $F_n = -q_s$ Flux

$F_n(+)-F_n(-) = -q_s$

3. $F_n = a(T - T_0)$ Convection

a - film coefficient

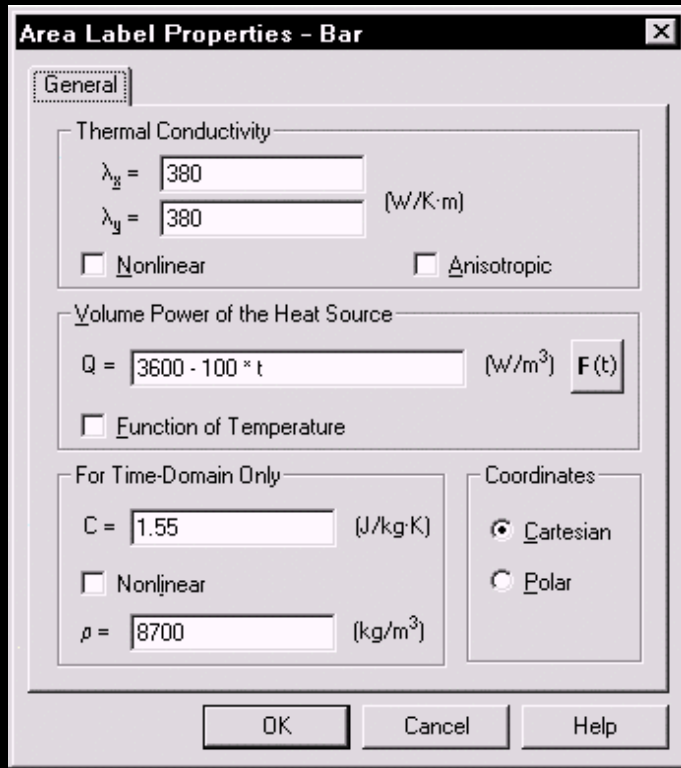
T_0 - temperature of contacting medium

4. $F_n = b.K_{sb}(T^4 - T_0^4)$ Radiation

K_{sb} - Stephan-Boltzmann constant;

b - emissivity coefficient

Boundary conditions & domain characterization



Area Label Properties - Bar

General

Thermal Conductivity

$\lambda_x = 380$ (W/K·m)

$\lambda_y = 380$ (W/K·m)

Nonlinear Anisotropic

Volume Power of the Heat Source

$Q = 3600 - 100 * t$ (W/m³) **F(t)**

Function of Temperature

For Time-Domain Only

$C = 1.55$ (J/kg·K)

Nonlinear

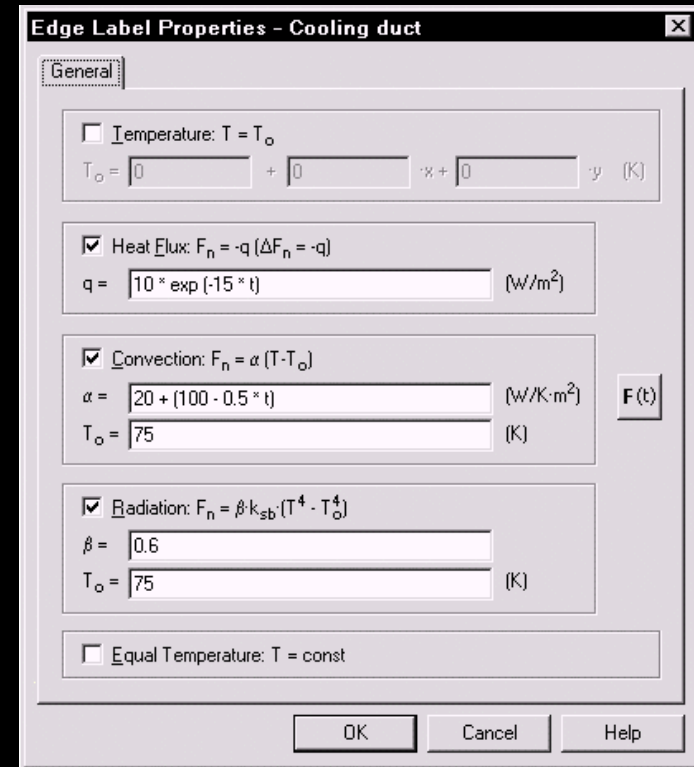
$\rho = 8700$ (kg/m³)

Coordinates

Cartesian Polar

OK Cancel Help

Volume element



Edge Label Properties - Cooling duct

General

Temperature: $T = T_o$

$T_o = 0 + 0 * x + 0 * y$ (K)

Heat Flux: $F_n = -q (\Delta F_n = -q)$

$q = 10 * \exp(-15 * t)$ (W/m²)

Convection: $F_n = \alpha (T - T_o)$

$\alpha = 20 + (100 - 0.5 * t)$ (W/K·m²) **F(t)**

$T_o = 75$ (K)

Radiation: $F_n = \beta * k_{sb} (T^4 - T_o^4)$

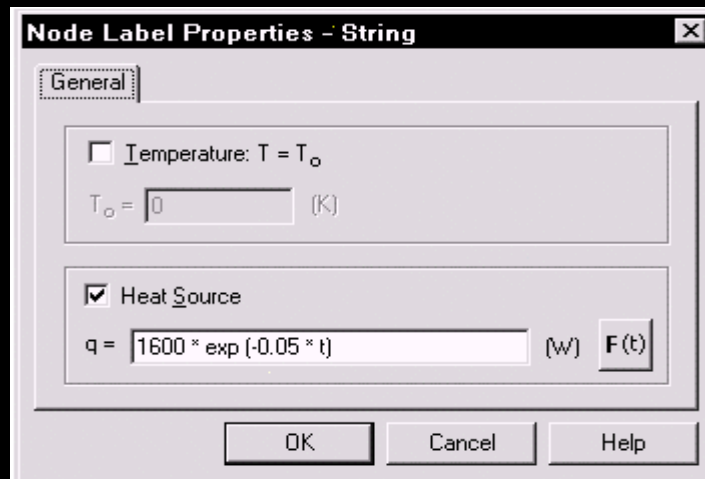
$\beta = 0.6$

$T_o = 75$ (K)

Equal Temperature: $T = \text{const}$

OK Cancel Help

Surface element



Node Label Properties - String

General

Temperature: $T = T_o$

$T_o = 0$ (K)

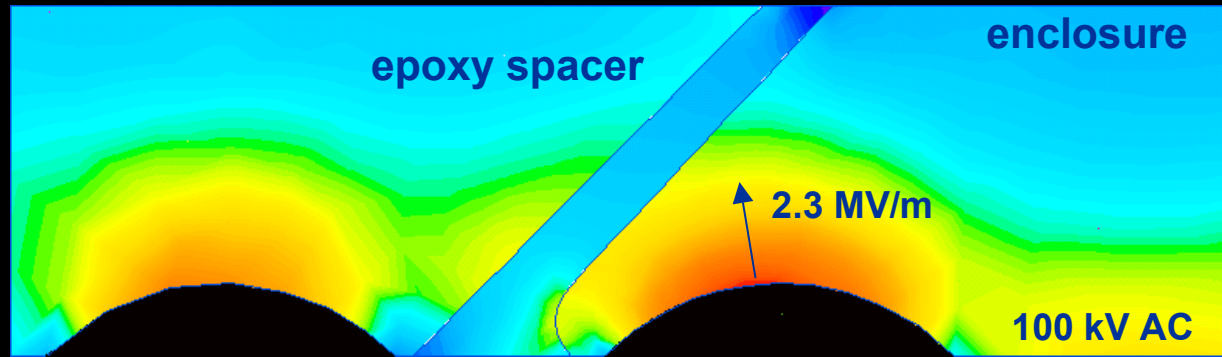
Heat Source

$q = 1600 * \exp(-0.05 * t)$ (W) **F(t)**

OK Cancel Help

Point-Source element

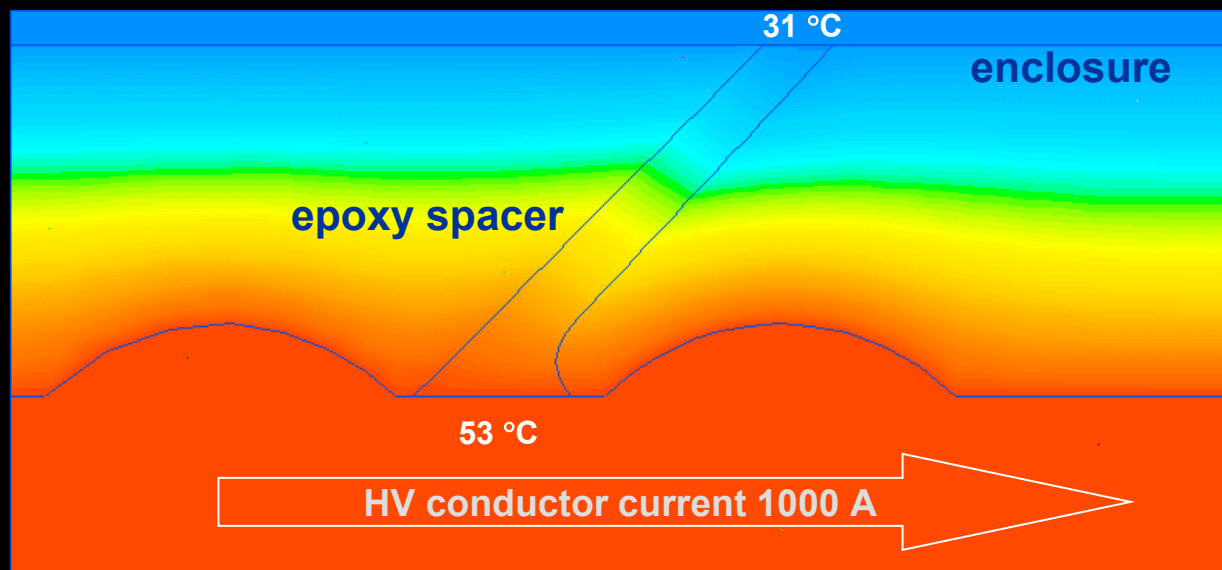
Coupling Problems solution for SF6 GIS 170 kV



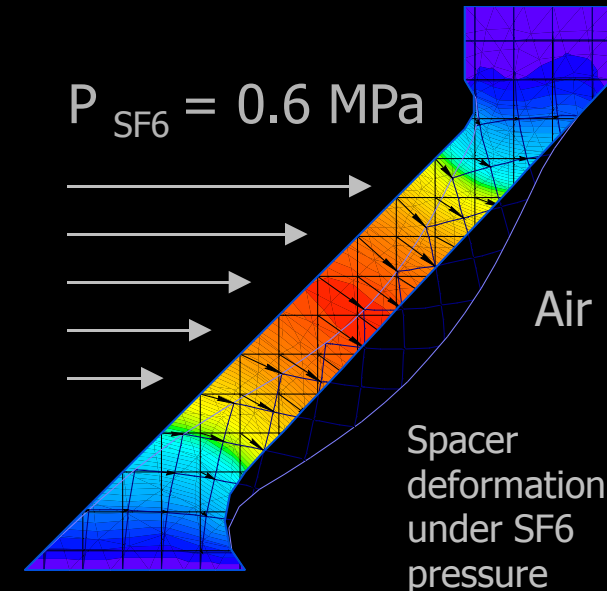
Electric Field distribution in GIS compartment



Joule losses distribution at central conductor 1000 A



Thermo-static field mapping in GIS compartment



Spacer deformation under SF6 pressure

SF6 GIS HT Model Parameters

1. SF6 Thermal Conductivity $\lambda_g = 0.0136$ W/m.K
2. Air Thermal Conductivity $\lambda_a = 0.026$ W/m.K
3. Epoxy Thermal Conductivity $\lambda_e \in (0.3-0.6)$ W/m.K
4. Aluminum Thermal Conductivity $\lambda_{al} \in (140-220)$ W/m.K
5. Copper Thermal Conductivity $\lambda_{cu} = 380$ W/m.K

6. Convection Parameters:

6.1. Internal SF6 space:

$$\varepsilon_k = 0.133(\text{Gr.Pr})^{0.28} \in (1.2 - 6.0)$$

$$10^3 < \text{Gr.Pr} < 10^6 \quad (\text{for SF6 GIS})$$

6.2. External Air space:

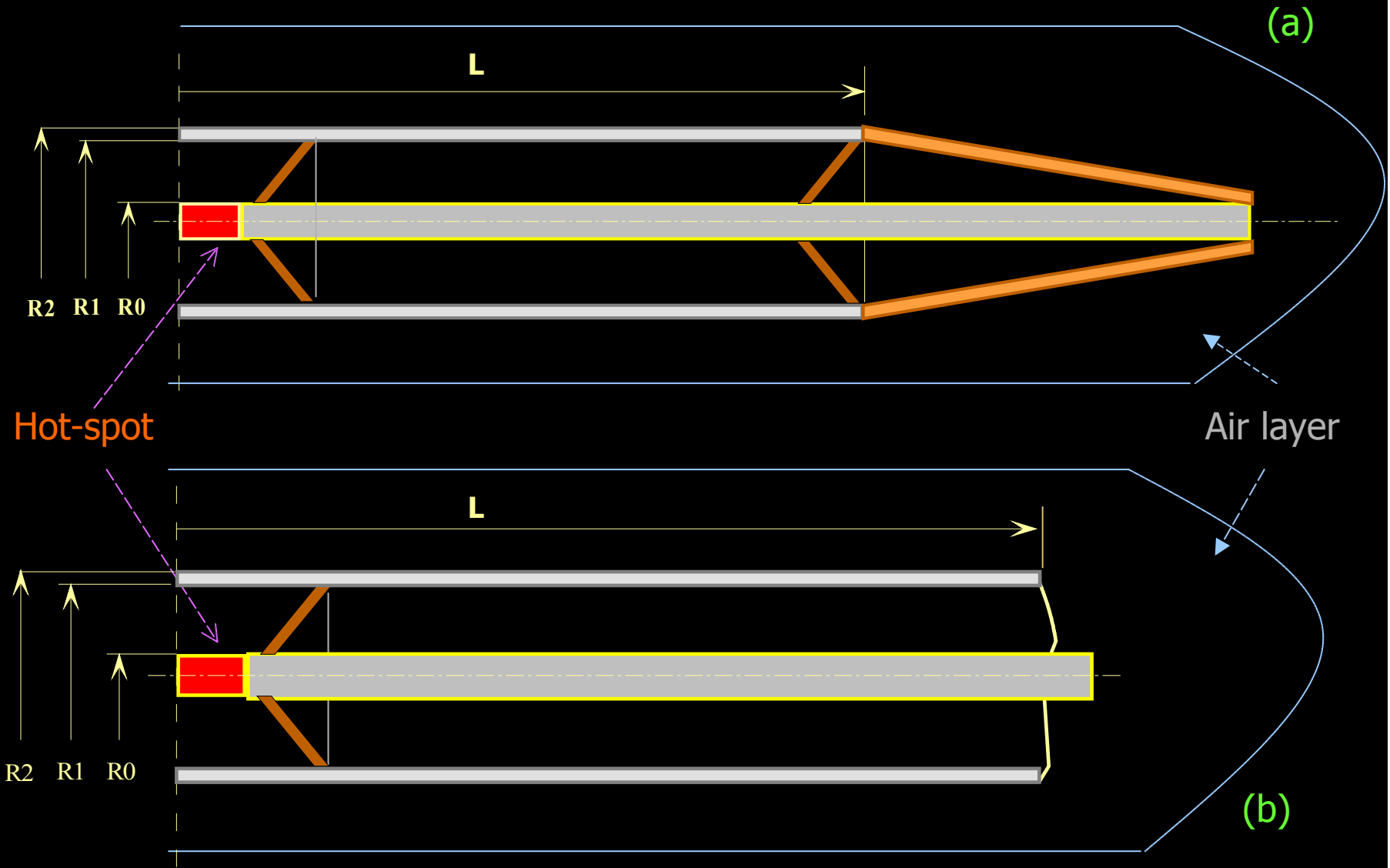
$$a_c \in (2-10) \text{ W/K.m}^2 ; T_0 \in (20-25^\circ\text{C})$$

7. Radiation Parameters:

$$\text{equivalent emissivity coefficient: } b_e \in (0.01 - 0.6)$$

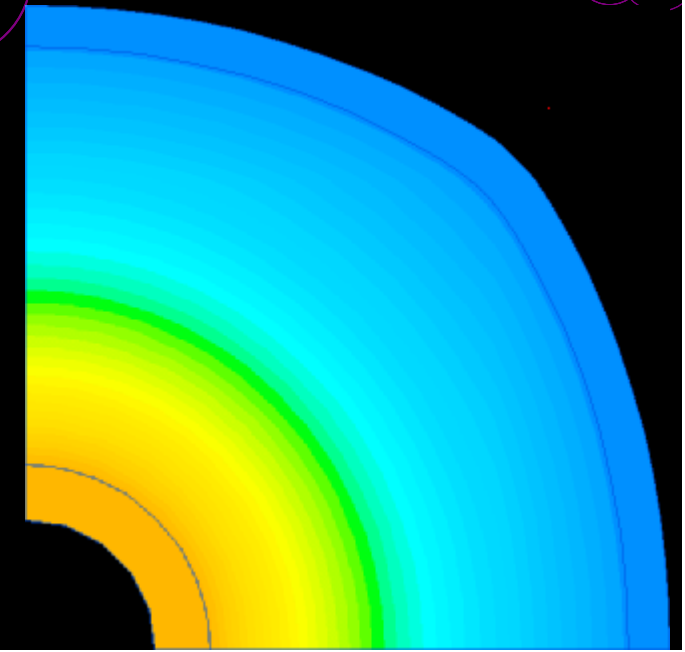
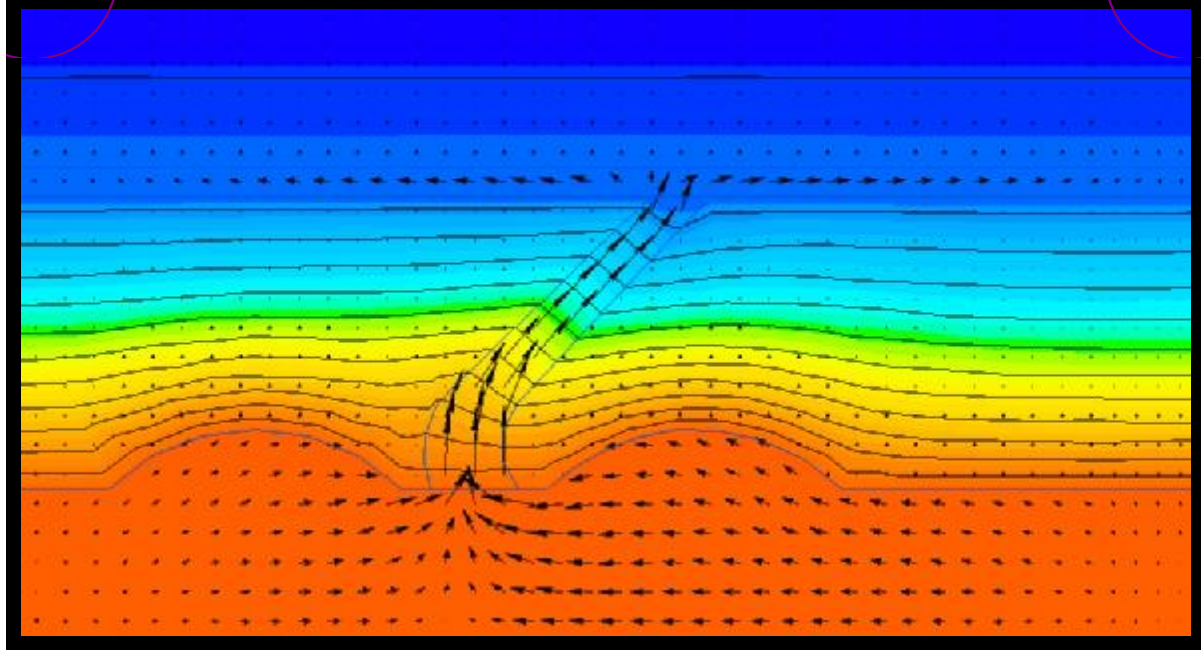
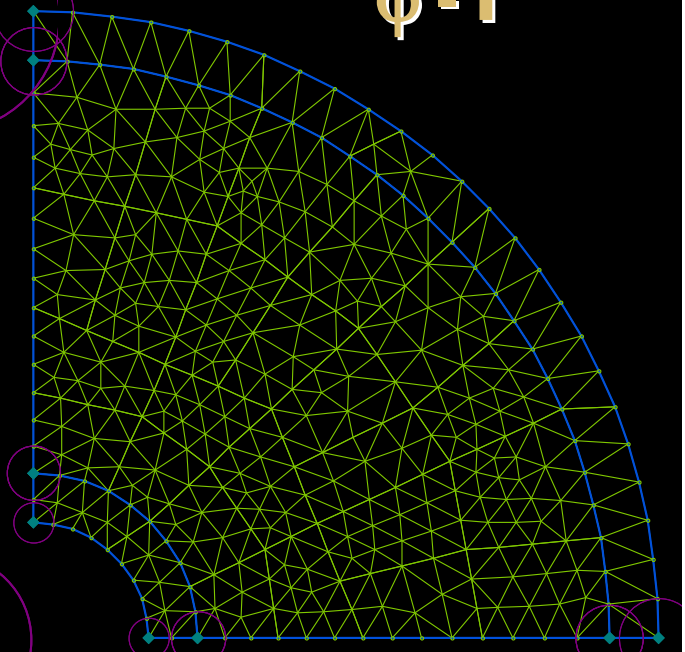
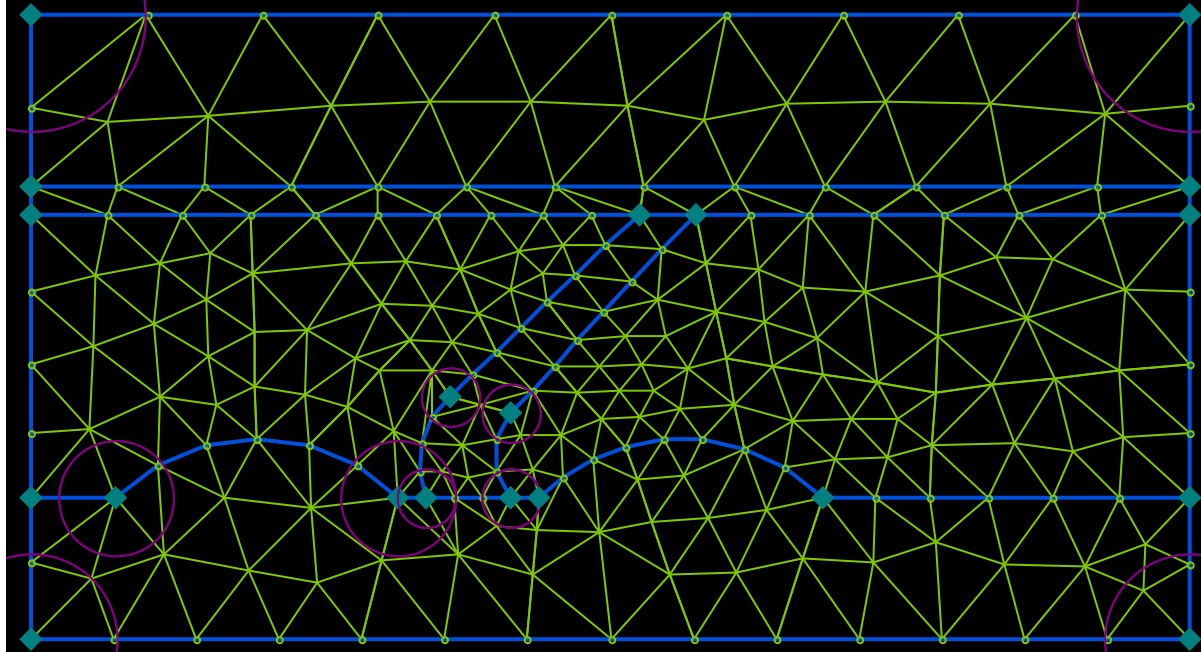
GIS Geometric Model examples

Symmetry Axis

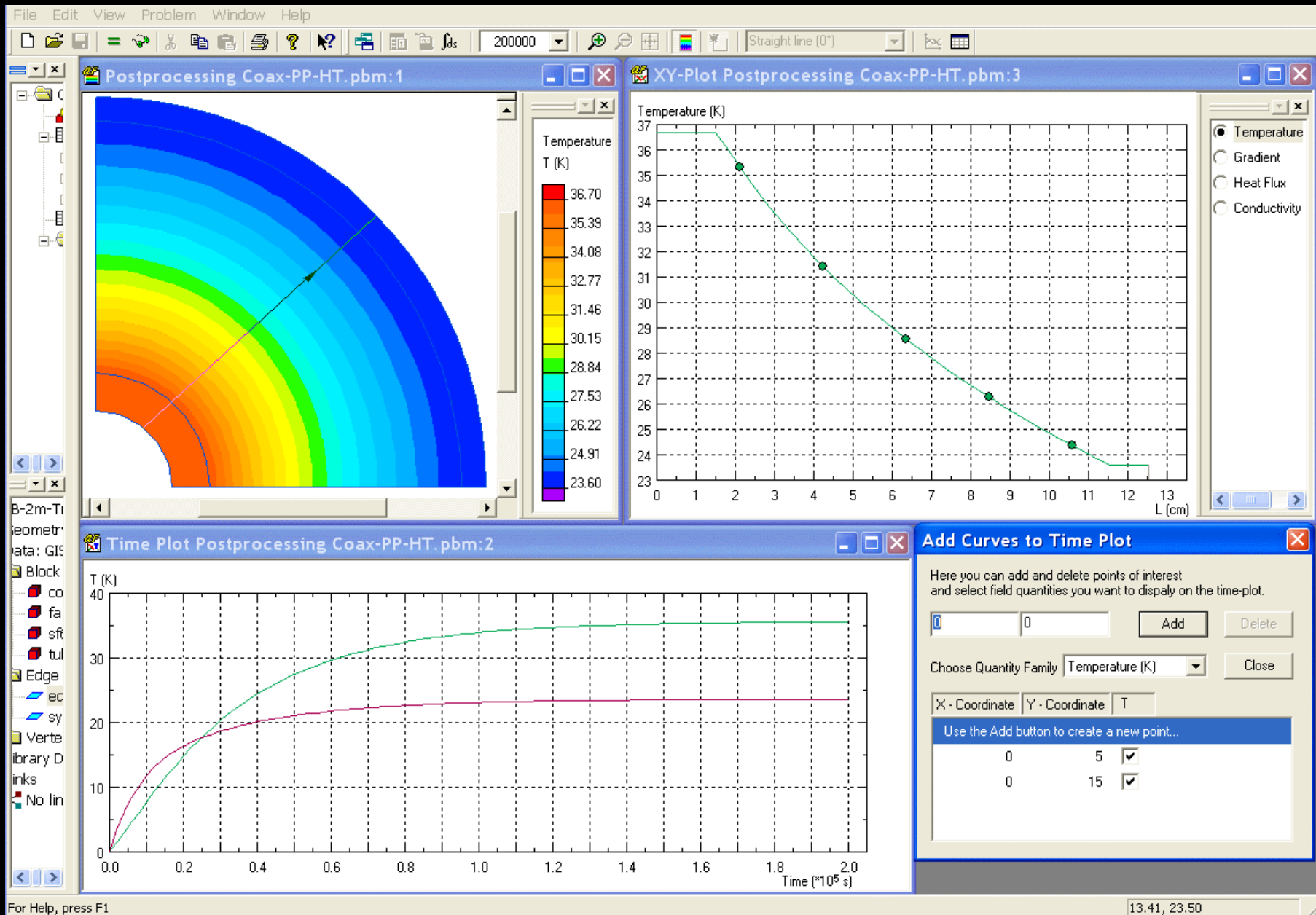


$r-z$ Geometric Models & results presentation

$\phi-r$

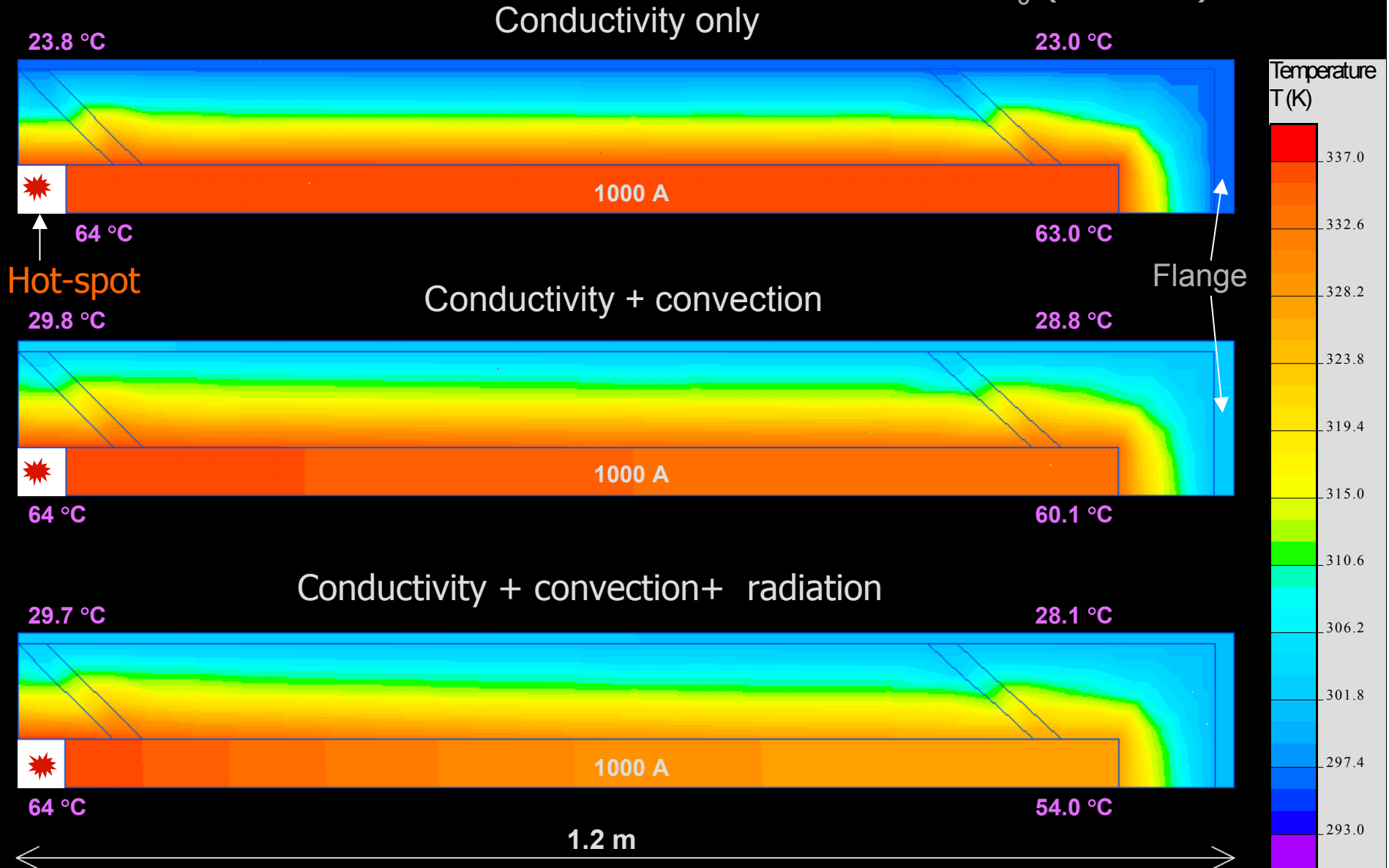


Geometric Models & results presentation



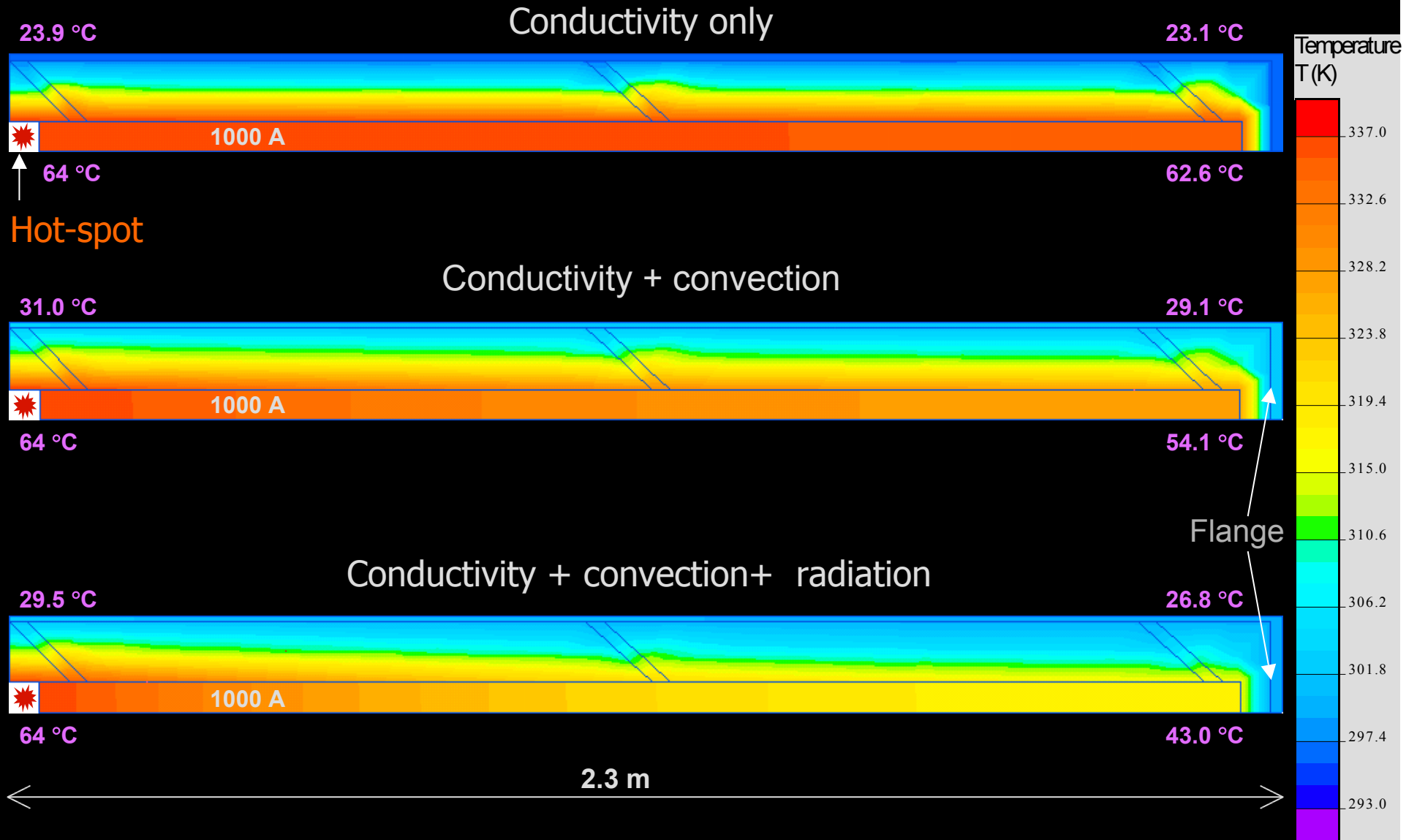
Thermal Field mapping for BB model 1.2 m

T_0 (ambient) = 20°C

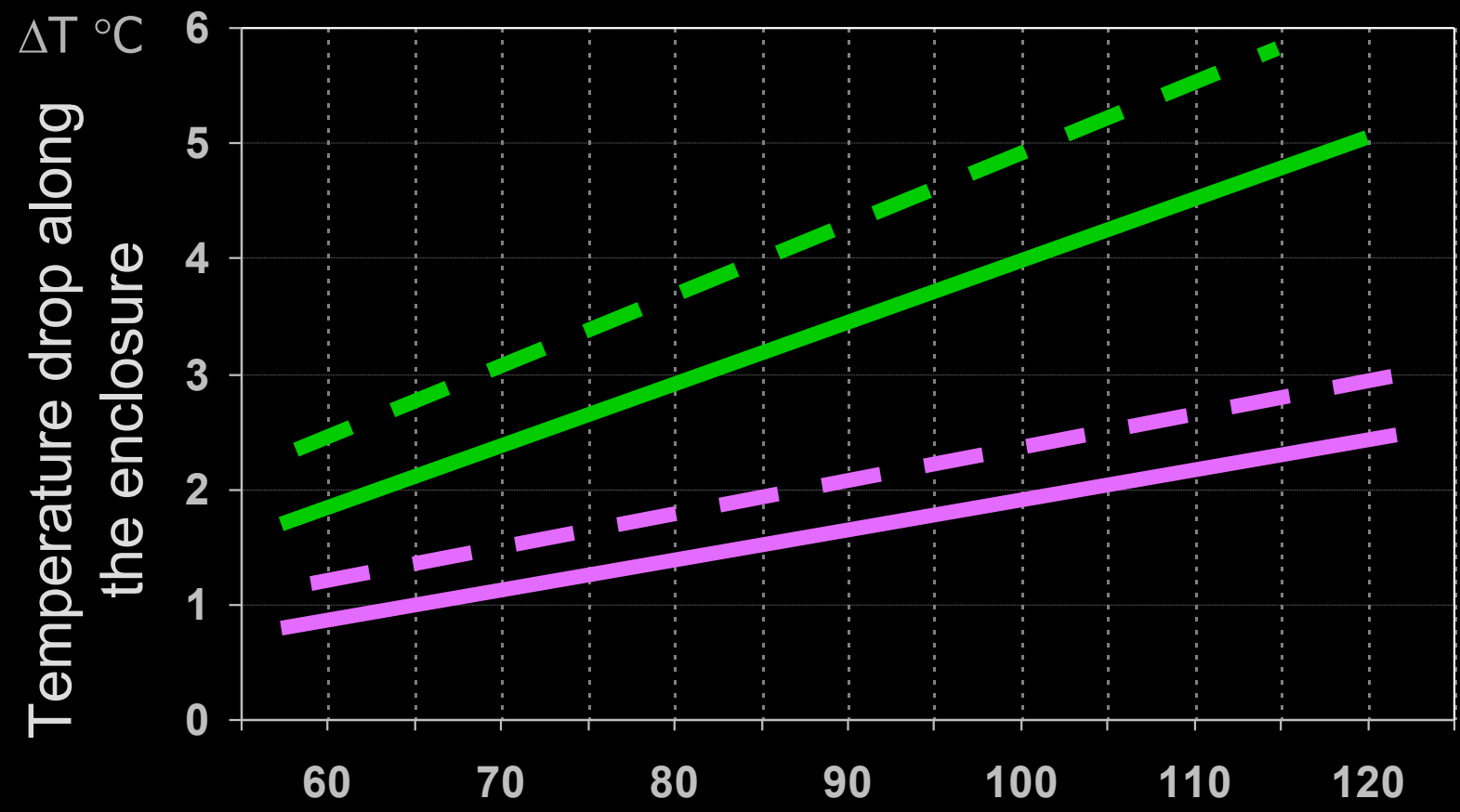


Thermal Field mapping for BB model 2.3 m

T_0 (ambient) = 20°C



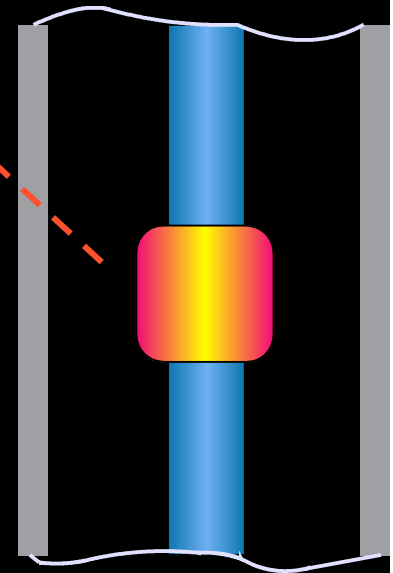
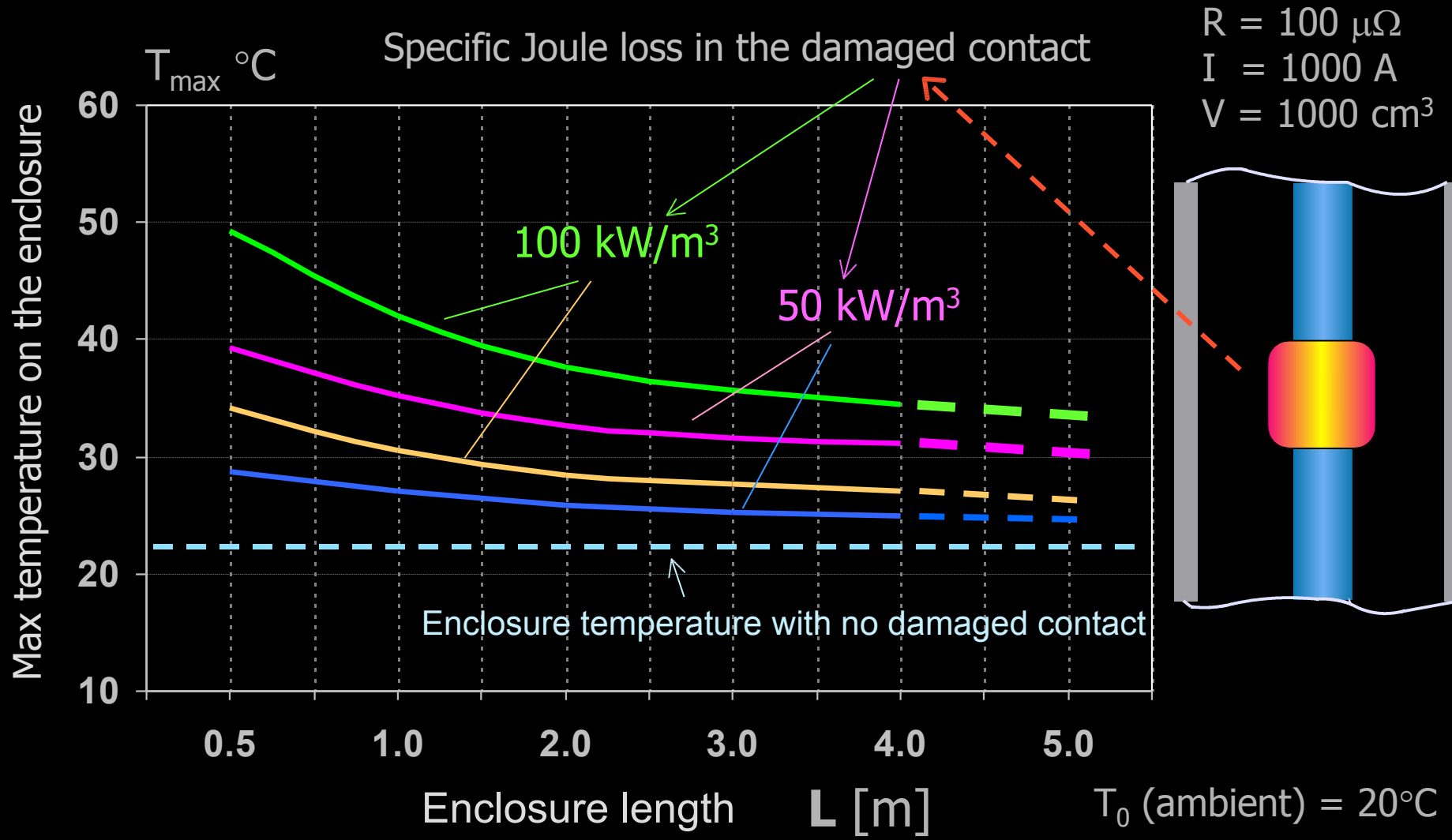
Enclosure overheating as a function of Hot-spot temperature



— BB length 1 m
— BB length 2 m
- - - Including radiation
- - - Excluding radiation

Hot-spot temperature T_{max} $^{\circ}\text{C}$

Enclosure overheating as a function of the BB length



Conductivity + convection █ Conductivity + convection+ radiation █
█ █

HT Transients for BB model

T_0 (ambient) = 20°C

T °C

